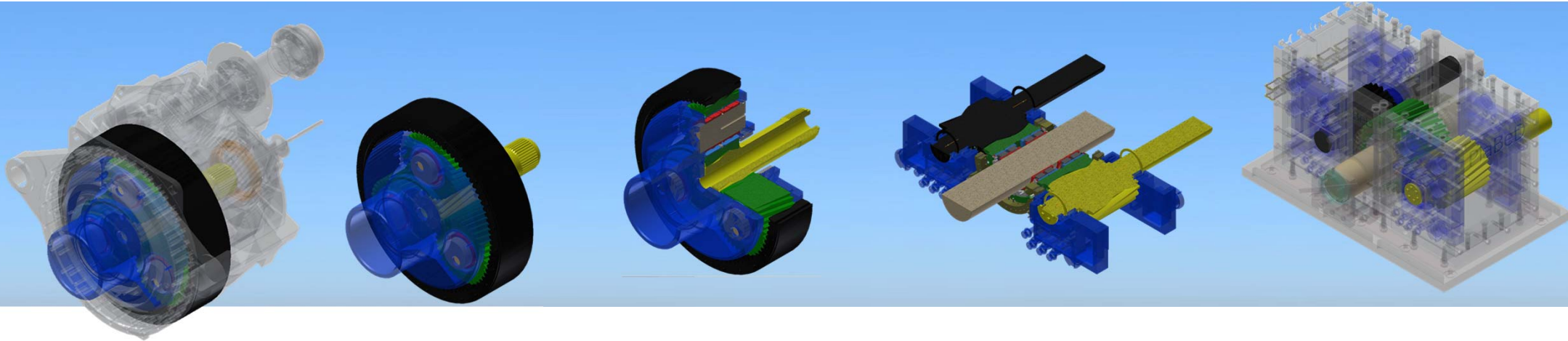




Development of an innovative component test bench for planetary bearings in wind turbine gearboxes

Project „PlaBeD“ – Planetary Bearing Design

Enhanced Design of Planetary Bearings in Gearboxes of Wind Turbines

09.11.2017

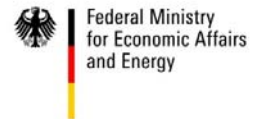
Valencia, Spain

Prof. Dr.-Ing. Georg Jacobs

Dipl.-Ing. Dennis Bosse

Felix M. Schlüter, M.Sc.

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Wind Power
Drives

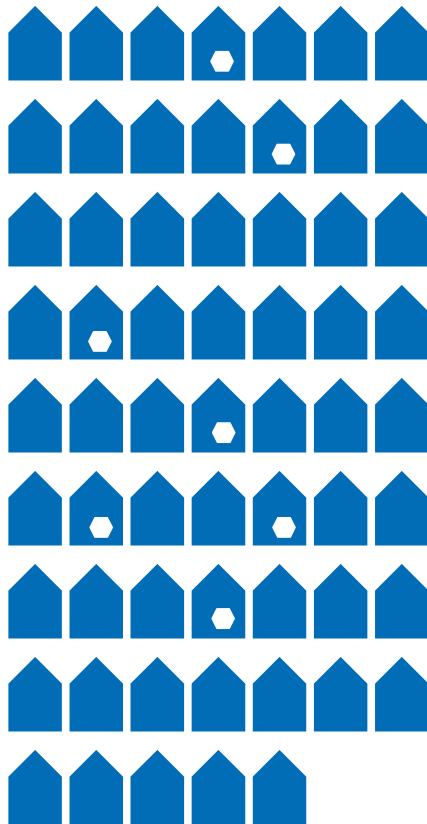
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Overview

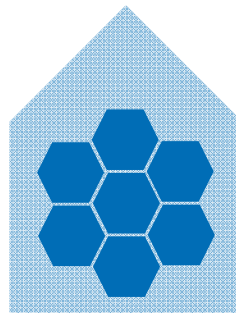
- 1 The Center for Wind Power Drives
- 2 Project „PlaDeD“
- 3 Test Bench Concept
- 4 Relevant deformations and misalignments
- 5 Detailed functionalities
- 6 Summary

Center for Wind Power Drives – Concept and Integration

RWTH



I³-Institute: Integrated Interdisciplinary Institute:
aerodynamics, control technology, electrical
and mechanical engineering under one umbrella



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Wind Power
Drives

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Hameyer, Monti, Jacobs, Schröder
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**Systemdesign
& Test**



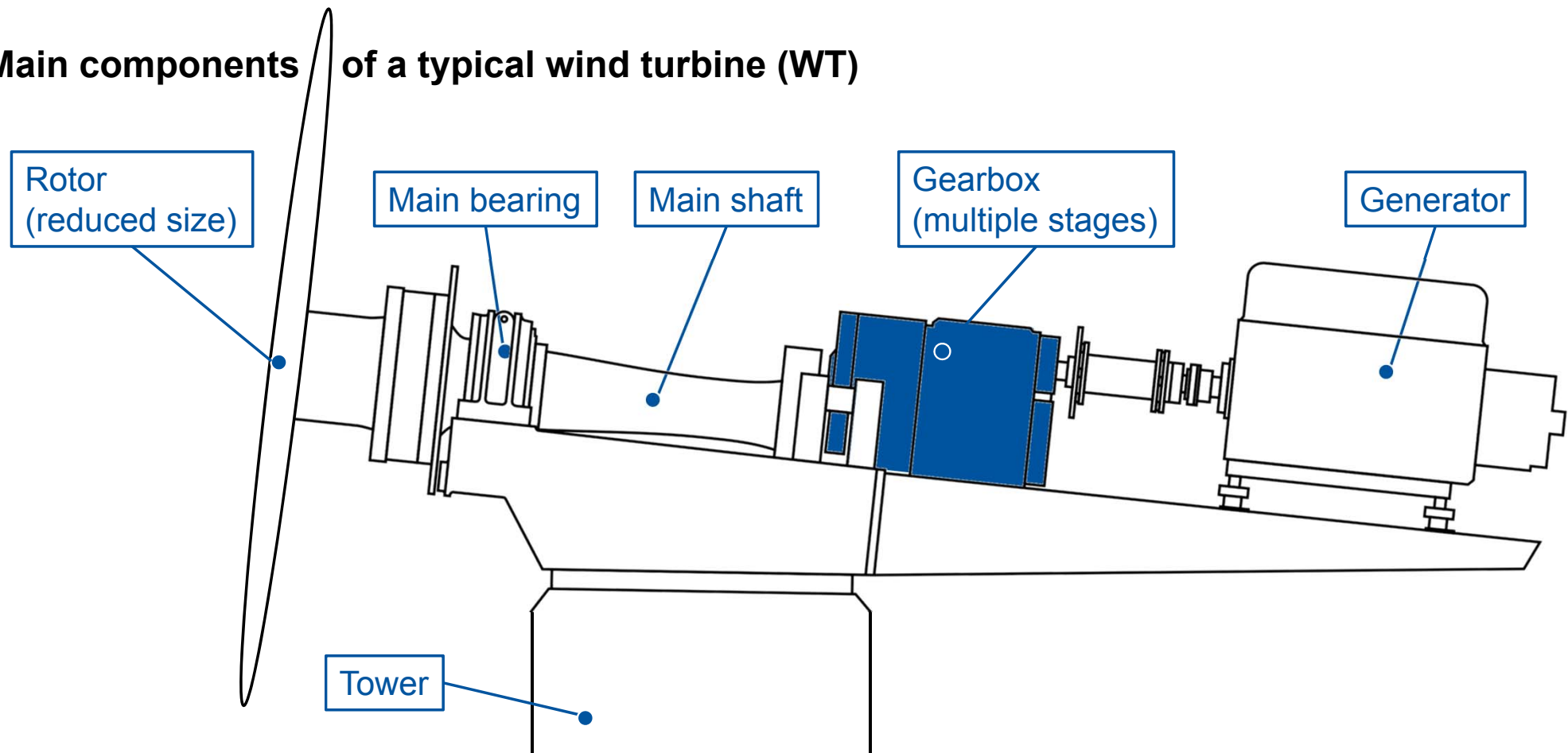
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Overview

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System overview „wind turbine“

Main components of a typical wind turbine (WT)



Motivation

Downtime of wind turbines / Gearbox failures

- About 90% of all WT have a planetary gearbox [1]
- Gearboxes cause about 60% of the downtime in WT [2]
- About 2/3 of these failures result from bearing damages [2]

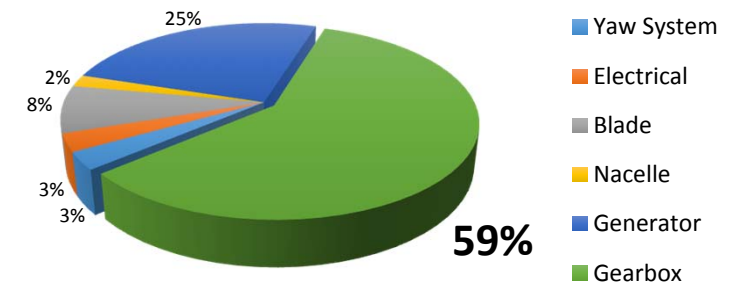
Planetary Bearings

- Highest loads in WT gearboxes appear in planetary bearings [3]
- Replacing a planetary bearing is very costly [4]

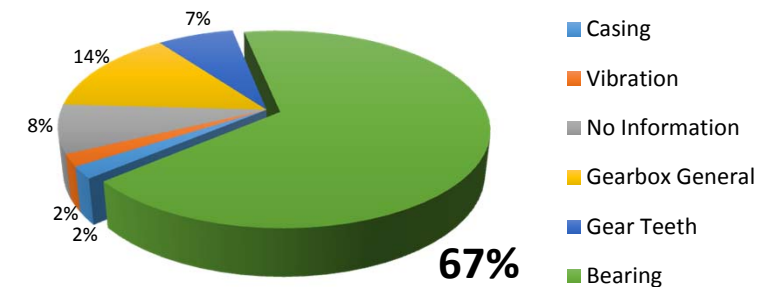
➡ More detailed understanding of failure mechanisms necessary

➡ Project „PlaBeD“ – Planetary Bearing Design
Enhanced Design of Planetary Bearings in Gearboxes of WT

Reasons for down time of WT [2]



Reasons for gearbox failures in WT [2]



Project „PlaBeD“ – Planetary Bearing Design

Problem

- Unknown deformations in planetary stage
- Unpredictable bearing load conditions

➡ Premature bearing failures

Goals

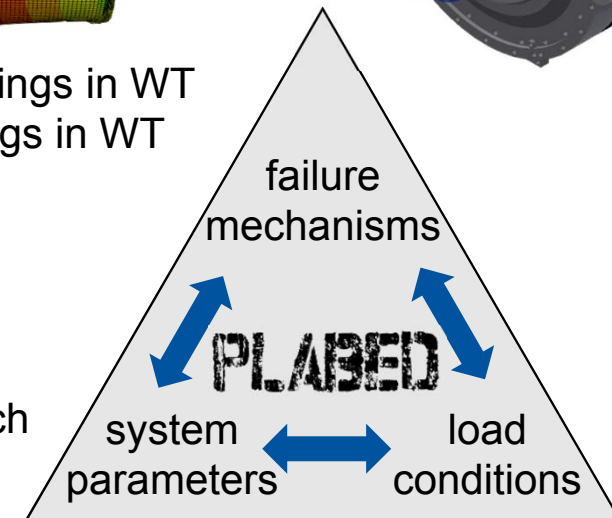
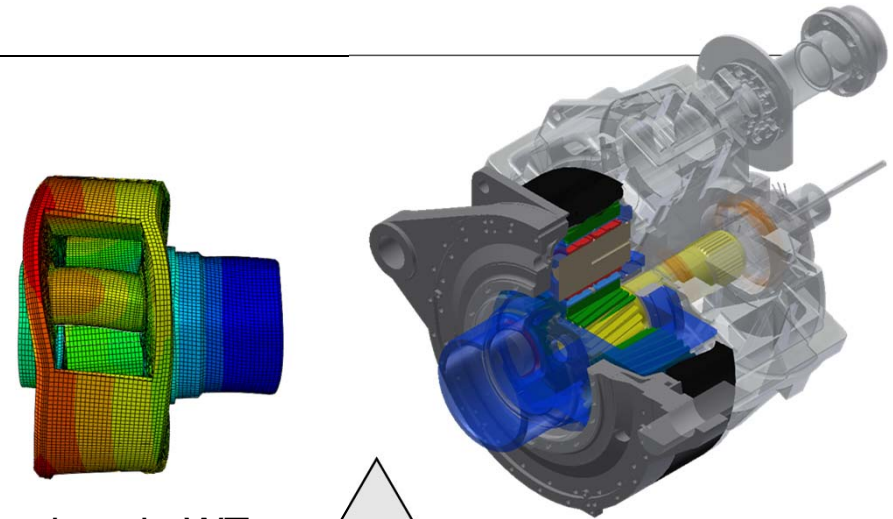
- Extend and validate current calculation methods for planetary bearings in WT
- Formulate an extended guideline for the design of planetary bearings in WT

➡ Increase economic efficiency of WT

Approach

- Test bench for planetary bearings
- Simulation of system deformations and transformation on test bench
- Reproduction of load conditions, evocation of failure mechanisms

➡ Profound system understanding



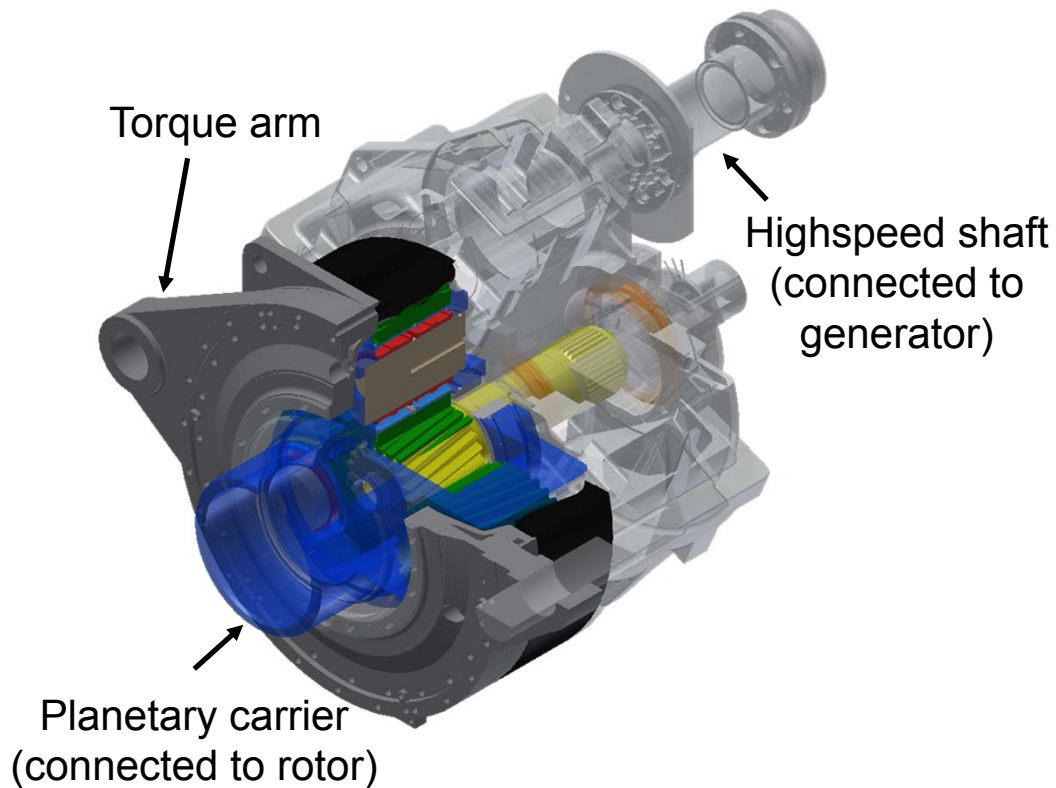
More information: <https://www.cwd.rwth-aachen.de/1/projects/plabed/>

Overview

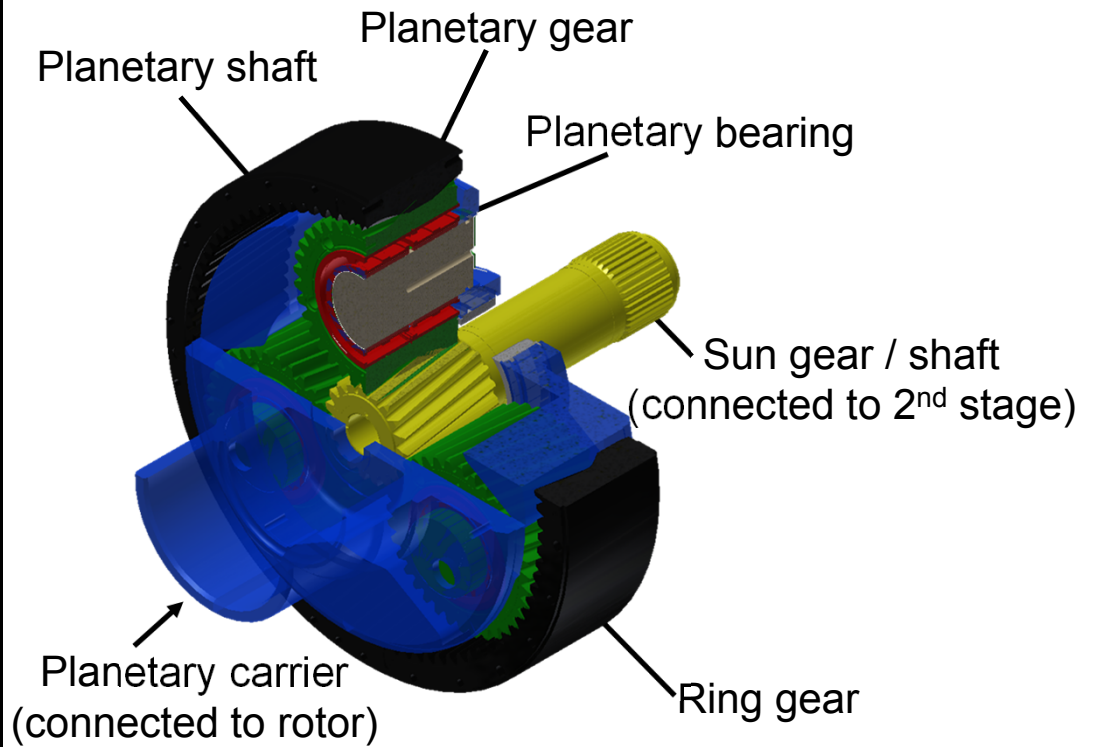
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System overview „gearbox“ and „planetary stage“

Gearbox



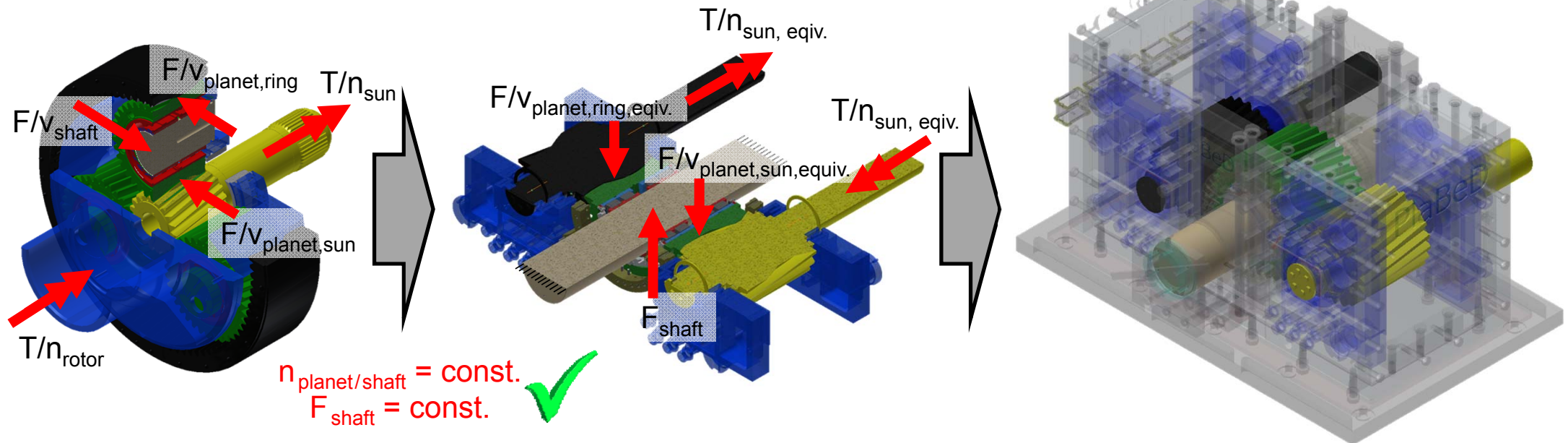
Planetary stage



Equivalent mechanical testing system

Transfer from WT gearbox to test bench

- Load conditions (torques and forces)
- Kinematic conditions (speeds and velocities)

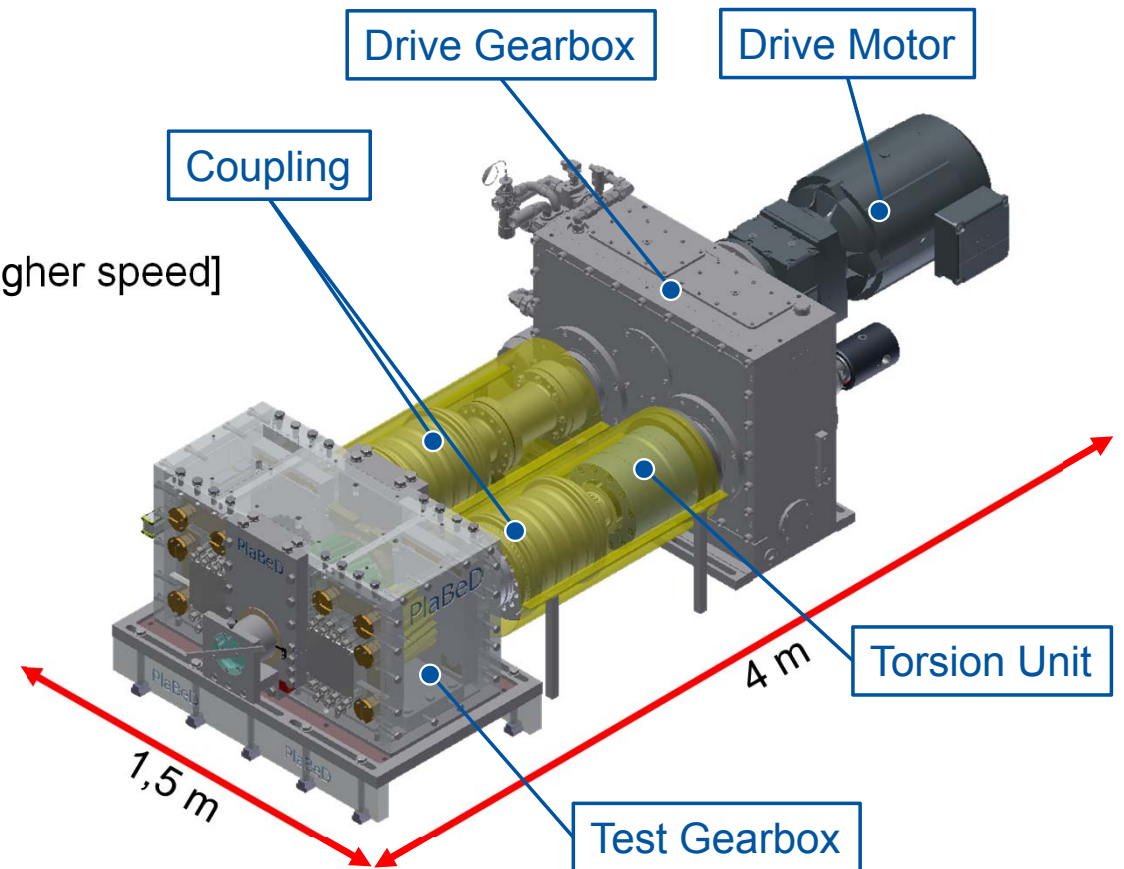
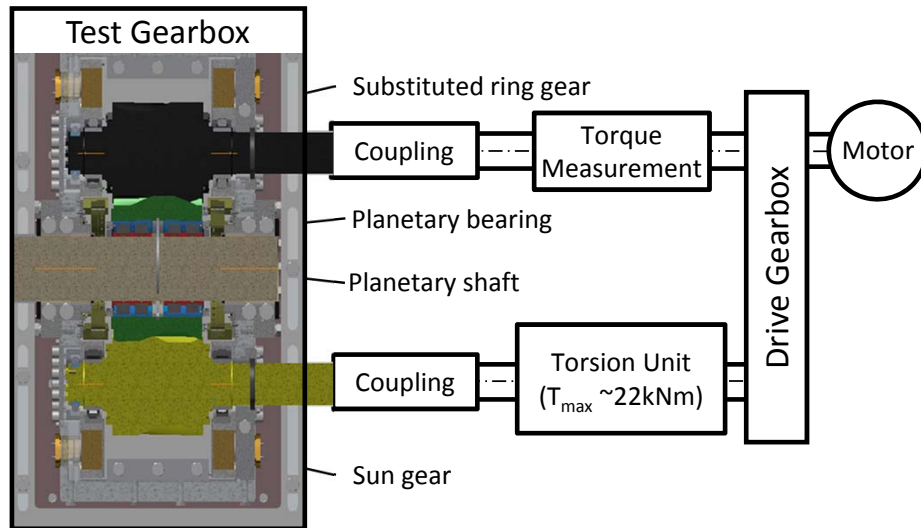


yellow = sun (shaft) green = planetary gear black = (substituted) ring gear blue = (substituted) planetary carrier

Test bench overview

Setup as mechanical bracing rig

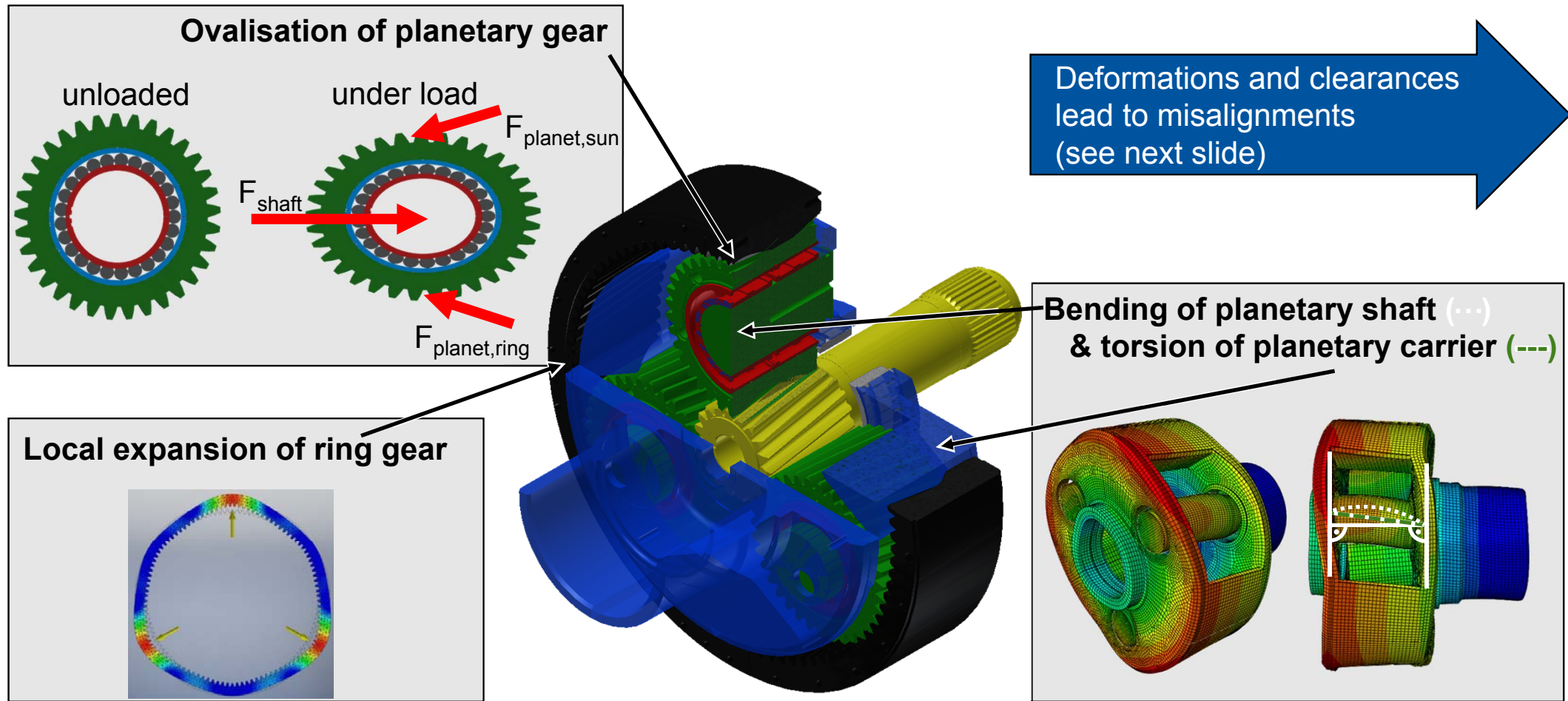
- Max. torque (sun): 22kNm
- Max. speed (planet bearing): 265 1/min
- Motor power: 55 kW
- Equiv. bracing power: 925 kW [$\sim 2,75$ MW WT at higher speed]



Overview

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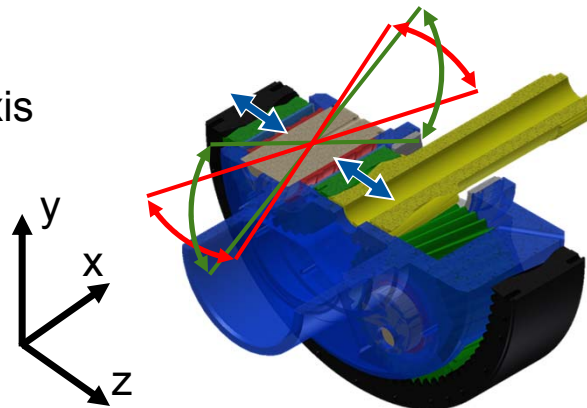
Relevant deformations in the planetary stage



Relevant misalignments in the planerary stage

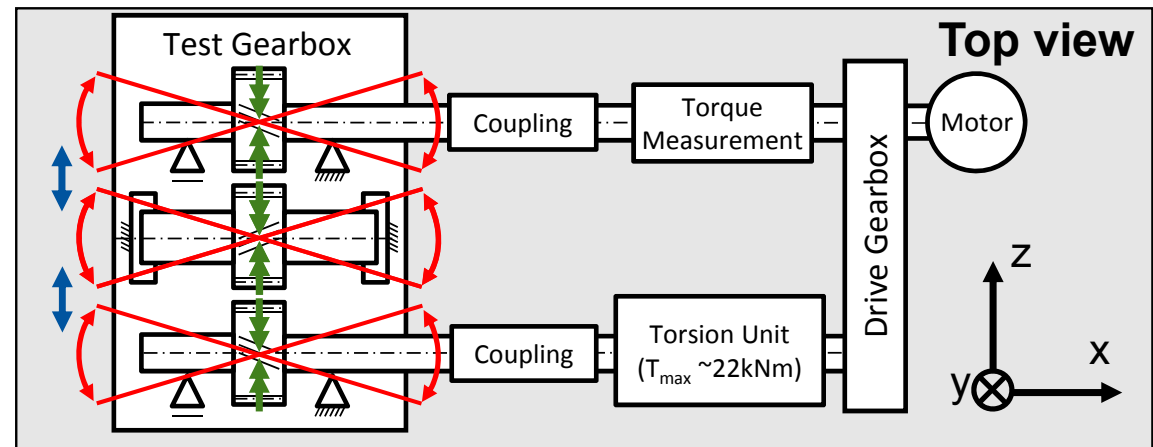
Misalignments

- Tilting of components around y- and z-axis
 - gears (sun, planetary gear, ring gear)
 - planetary carrier
- Change of axis distances
 - sun – planetary gear
 - ring gear – planetary gear



Transfer to test bench

- Independent active adjustment of all relevant angles and distances on test bench possible
 - axis distances
 - gear alignment angles
- Tilting of planetary gear transferred to sun and ring gear



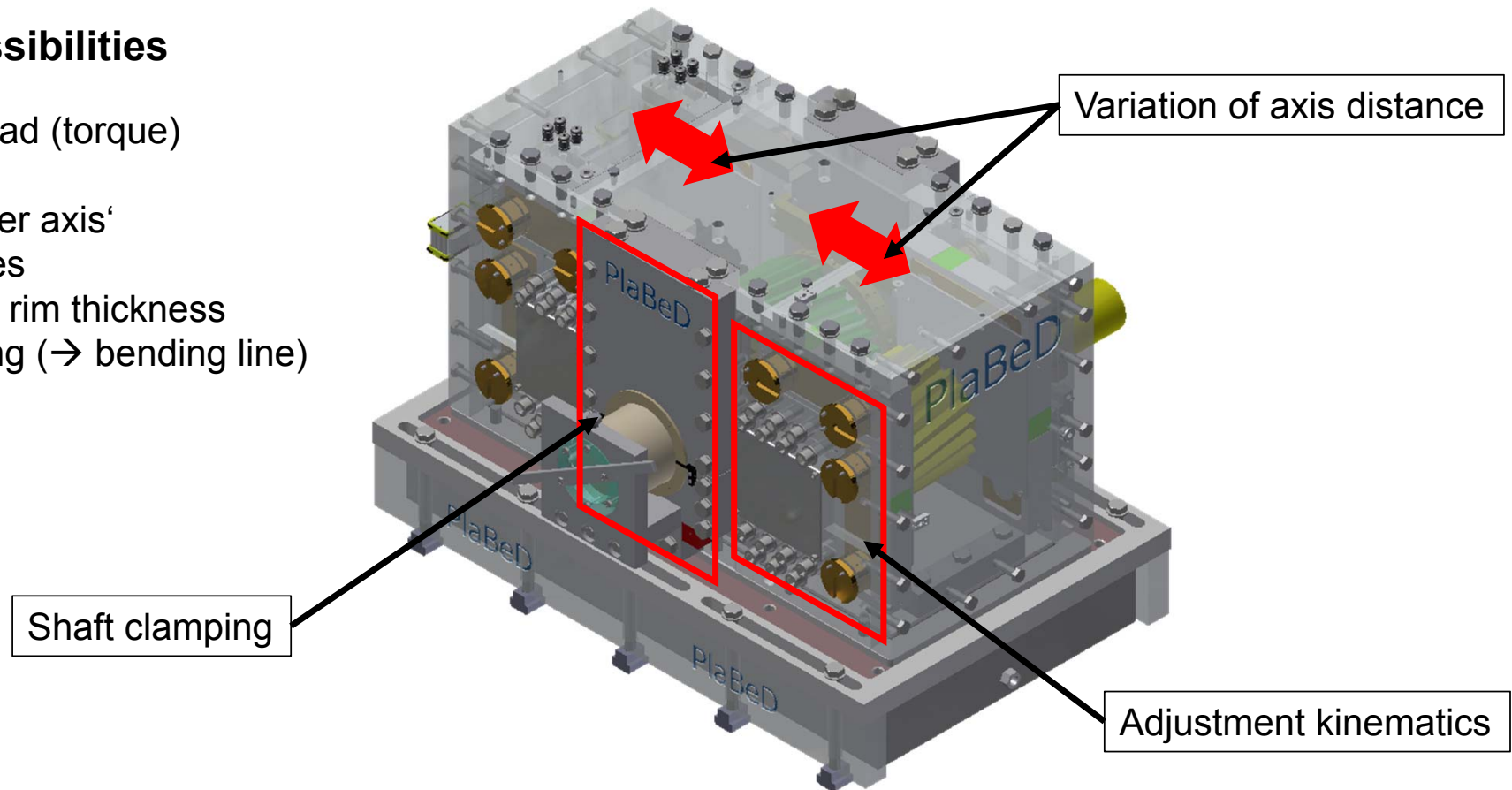
Overview

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Detailed functionalities – Overview

Variation possibilities

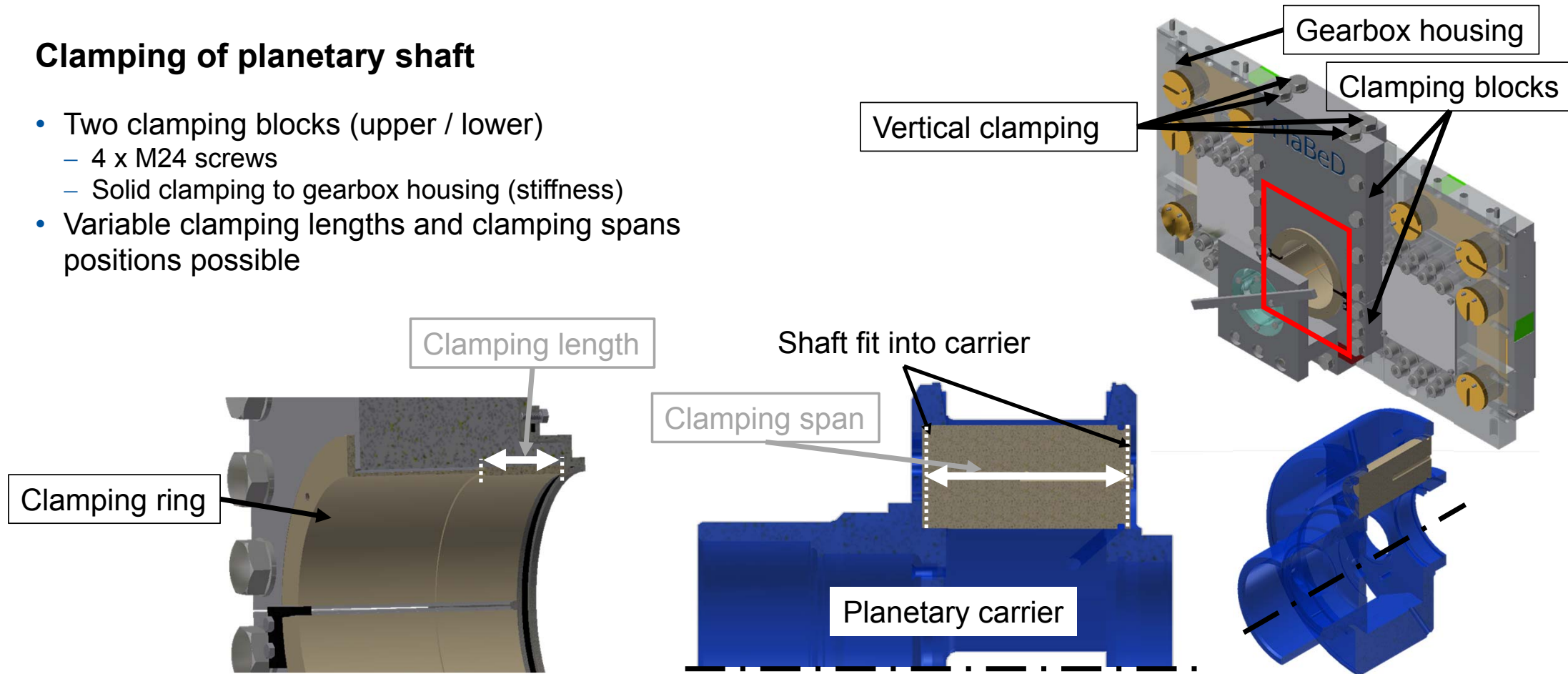
- Speed and load (torque)
- Temperature
- Angles of outer axis'
- Axis' distances
- Relative gear rim thickness
- Shaft clamping (→ bending line)



Detailed functionalities (1)

Clamping of planetary shaft

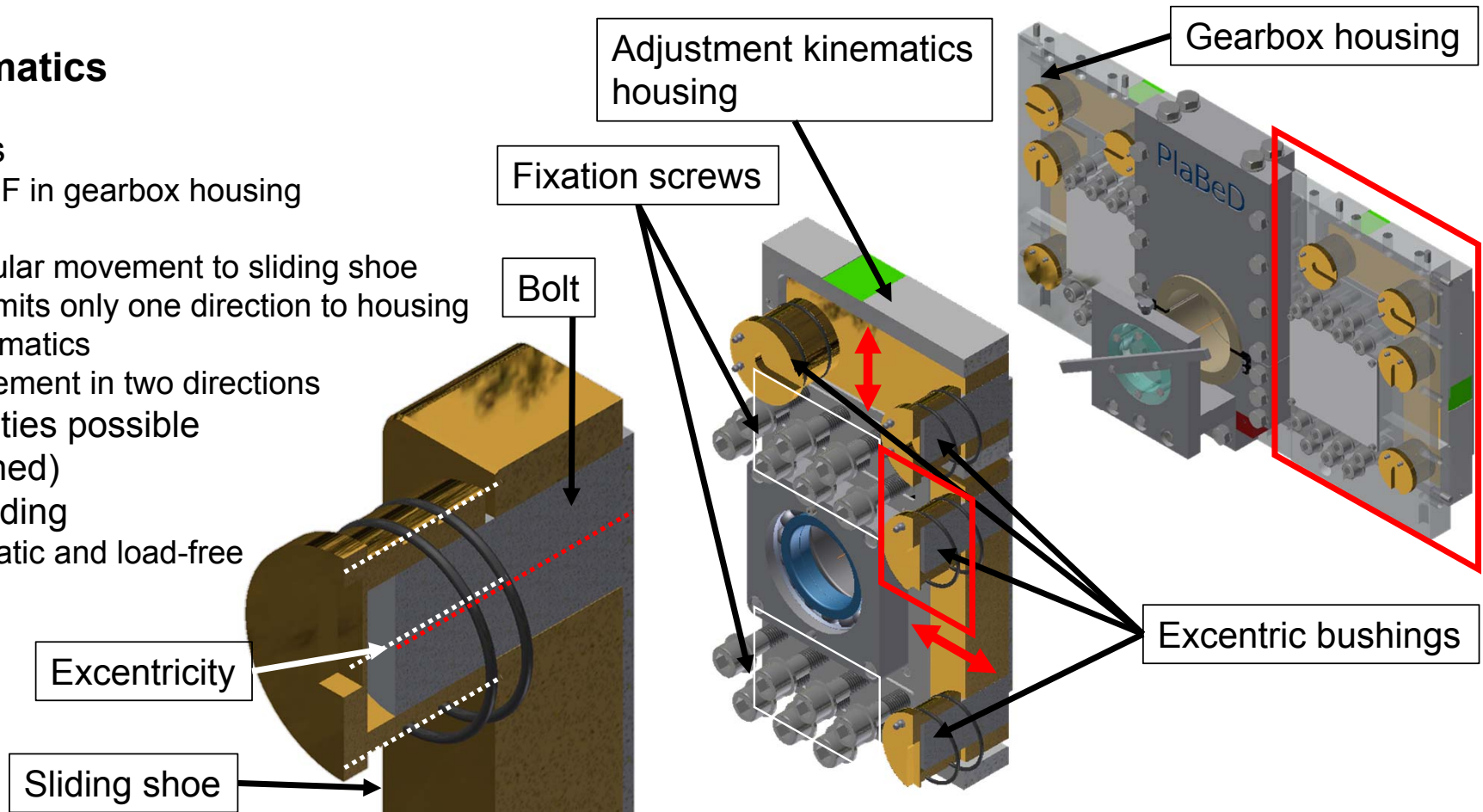
- Two clamping blocks (upper / lower)
 - 4 x M24 screws
 - Solid clamping to gearbox housing (stiffness)
- Variable clamping lengths and clamping spans positions possible



Detailed functionalities (2)

Adjustment kinematics

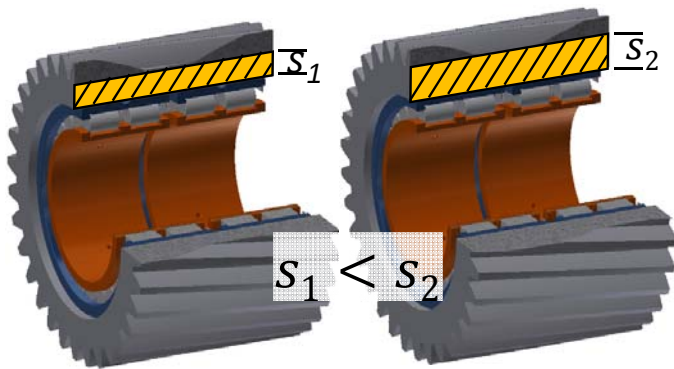
- Excentric bushings
 - Only rotational DOF in gearbox housing
 - Excentric bore
 - Bolt transmits circular movement to sliding shoe
 - Sliding shoe transmits only one direction to housing of adjustment kinematics→ Independent movement in two directions
- Different excentricities possible (~25-200 μm planned)
- Fixation before loading
→ Movement only static and load-free



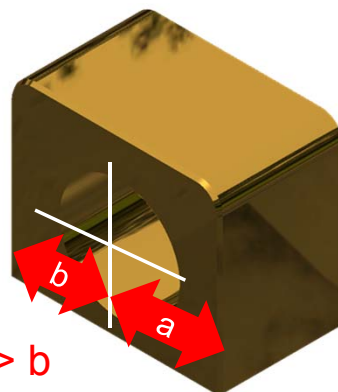
Detailed functionalities (3)

Variation of axis distance

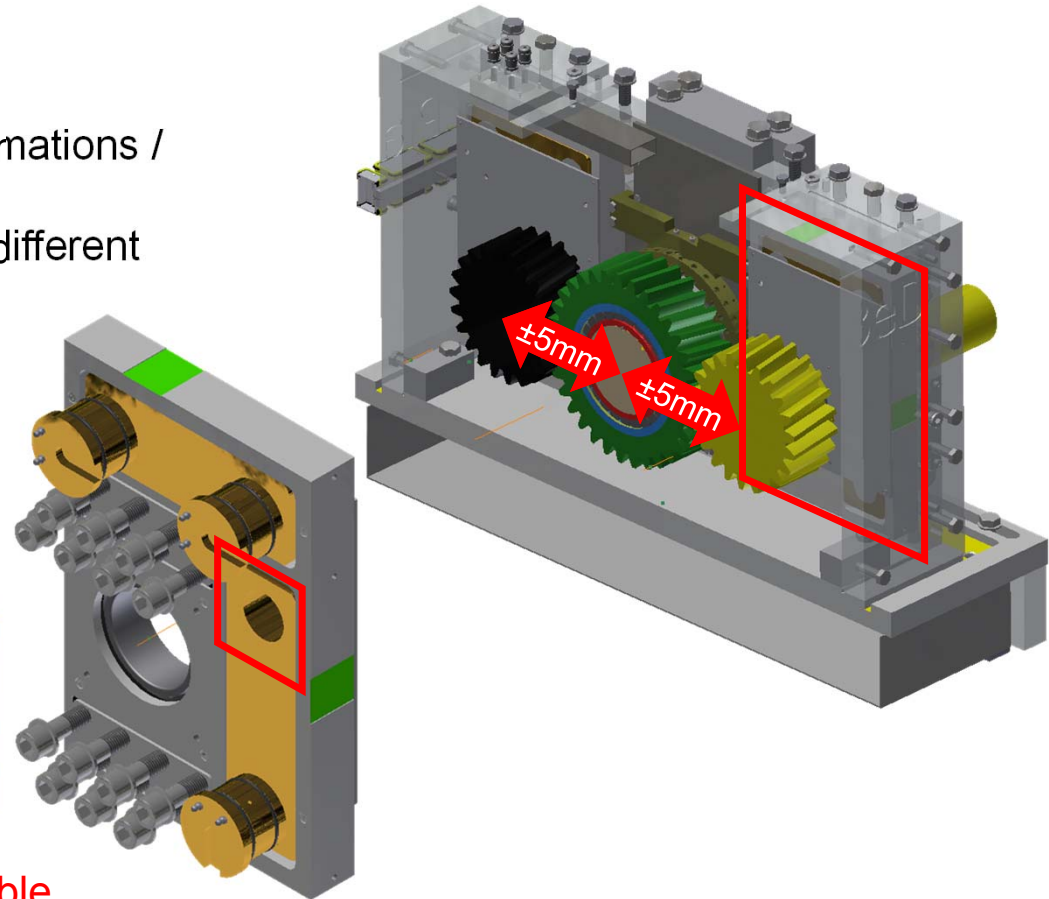
- In range of **μm continuously** to reproduce carrier deformations / local ring gear expansion (adjustment kinematics)
- In range of **mm stepwise** to allow planetary gears with different „gear rim thickness“
 - Different gear rim elasticity
 - Different influence on planetary bearing (tooth bending, ovalisation of planetary gear)



Gears with different rim thickness



$a > b$
Both directions possible



Overview

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Summary

- Motivation
 - Premature failures in planetary bearings in WT
 - Detailed investigation of failure mechanisms necessary
- Test Bench Concept
 - Equivalent mechanical testing system
 - One instead of three planets
 - Mechanical bracing rig
 - Testing equivalent of 2,8 MW WT with 55 kW motor
- Deformations and Misalignments
 - Simulation of system deformations
 - Misalignments resulting from deformations
 - Transfer of misalignments on test bench kinematics
- Detailed functionalities
 - Variation of speed, torque and temperature
 - Variation of outer axis' angles and axis' distances
 - Manipulation of planetary shaft's bending line

Motivation

Downtime of wind turbines / Gearbox failures

- About 90% of all WT have a planetary gearbox [1]
- Gearboxes cause about 60% of the downtime in WT [2]
- About 2/3 of these failures result from bearing damages [2]

Reasons for down time of WT [2]

Planetary Bearings

- Highest loads in gearbox
- Replacing a planetary bearing is very expensive

More detailed investigation of failure mechanisms necessary

Equivalent mechanical testing system

Transfer from WT gearbox to test bench

- Load conditions (torques and forces)
- Kinematic conditions (speeds and velocities)

Relevant deformations in the planetary stage

Ovalisation of planetary gear

under load

Deformations and clearances lead to misalignments (see next slide)

Relevant misalignments in the planetary stage

Misalignments

- Tilting of components around y- and z-axis
 - planetary carrier
 - planetary gear
- Change of axis distances
 - sun – planetary gear
 - ring gear – planetary gear

Transfer to test bench

- Independent active angles and distances
- Tilting of planetary ring gear

Detailed functionalities (2)

Adjustment kinematics

- Excentric bushing
 - Only rotational DOF in gearbox housing
 - Excentric bore
 - Bolt transmits circular movement to sliding shoe
 - Sliding shoe transmits only one direction to housing of adjustment kinematics
- Independent movement in two directions
- Different excentricities possible
 - (~25-200 µm planned)
- Only static and load-free movement
- possible
- Fixation before loading

Labels in diagram: Adjustment kinematics housing, Gearbox housing, Excentric bushing, Sliding shoe, Bolt, Excentricity.

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Thank you for your attention

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Sources:

- [1]: Crowther, A.: Sources of time-varying contact stress and misalignments in wind turbine planetary sets, 2010
- [2]: Scott, K.G.; Infield, D.: Effects of Extreme and Transient Loads on Wind Turbine Drive Trains, AIAA Technical Conferences, Boston, 2013
- [3]: Thalmann, D.; Duskunovic, F.: Lager für Leistung, K-Magazin, Ausgabe 5/6 2009
- [4]: Bozan, C.; Cugno, S.: Developments and trends in wind turbine rolling bearings, Organi di trasmissione, Issue 5, 2010